

## Direct Forcing methods in Trio\_U theory and applications

M. Belliard, Clarisse Fournier, Y. Gorsse, C. Introini, B. Mathieu, B. Zope

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#### DE LA RECHERCHE À L'INDUSTRIE

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Direct Forcing methods in Trio\_U: theory and applications

Trio\_U Meeting | MICHEL BELLIARD

joint work with CLARISSE FOURNIER, YANNICK GORSSE, CLÉMENT INTROINI, BENOIT MATHIEU and BÉATRICE ZOPÉ

CEA/DEN

28/01/2015

Trio\_U Meeting INSTN, Saclay

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#### Introduction

- Theoretical developments
- Numerical applications
- Conclusions and Perspectives

## Introduction

## Cea The Fictitious Domain approach

#### Complex flow regimes combined with time-varying geometries

- Fluid-Structure Interactions in the nuclear components
  - $\star$  Flow-induced vibrations  $\rightarrow$  possible fuel-rod or U-tube damages
  - \* Fast FSI computations with large deformations
- Vitrification process for the storage of radioactive wastes
  - ★ Multiphase multicomponent flow at high temperature (gas bubbles, molten glass)
  - \* Static (vessel structure, measuring apparatus), moving (mechanical stirrer) bodies



## The Fictitious Domain approach

#### Complex flow regimes combined with time-varying geometries

#### Fluid-Structure Interactions in the nuclear components

- $\star$  Flow-induced vibrations  $\rightarrow$  possible fuel-rod or U-tube damages
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#### Fictitious Domain approach

Pioneer's work ([Saul'ev 1963])

- original domain embedded in a geometrically bigger and simply-shaped domain : fictitious domain
- computat. domain = fictitious domain
- Cartesian mesh  $\Rightarrow$  fast linear solvers,

good numerical properties, easy

multiresolution techniques



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Cea The Direct Forcing method

#### Fluid governing equations

$$\begin{cases} \frac{\partial \underline{u}}{\partial t} & +\nabla \cdot (\underline{u} \otimes \underline{u}) + \frac{1}{\rho} \underline{\nabla} p - \nu \nabla^2 \underline{u} = \underline{g} \quad \text{and} \quad \nabla \cdot \underline{u} = 0 \quad \text{in } \Omega_F \\ \underline{u} & = \underline{u}_D \text{ on } \partial \Omega_F \end{cases}$$



Cea The Direct Forcing method

#### Fluid governing equations



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TO LA RECHERCHE À CINEDATA



#### A test case from Seo & Mittal, JCP 230 (2011)

$$\begin{array}{lll} x_c(t) &=& x_c(0) + X_0(1 - \cos(2\pi t_0 t)) \\ y_c(t) &=& y_c(0) \end{array}$$

Fluid at rest Drag coefficient  $C_T = C_P + C_\nu$  $C_{DF} = \frac{\int \int \int_{\Omega_S} \rho \underline{E}_{DF} \cdot \underline{e}_x d\Omega}{\underline{E}_{ref} \cdot \underline{e}_x}$ 

х

or 14 presented Automotive

## Cea SFO

#### A test case from Seo & Mittal, JCP 230 (2011)

$$\begin{array}{lll} x_c(t) &=& x_c(0) + X_0(1 - \cos(2\pi f_0 t)) \\ y_c(t) &=& y_c(0) \end{array}$$



TO LA RECHERCHE À CINEDATI



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Forcing term is discontinuous in time ....

$$\int_{\Omega_{s,h}^{n+1}} \underline{E}_{DF}^{n+1} d\Omega - \int_{\Omega_{s,h}^{n}} \underline{E}_{DF}^{n} d\Omega = \mathcal{O}(1) + \mathcal{O}(\Delta x) + \mathcal{O}(\frac{\Delta x^{2*}}{\Delta t})^{*} : 3 \text{ for the linear model}$$

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Theoretical developments

The Penalized Direct Forcing method [Belliard, Fournier, C.R. Acad. Sci. Paris, 2010] & [Introini et al., Computers and Fluids, 2014]

A combination of the Direct Forcing method [Mohd-Yusof, *CTR An. R. Briefs*, 1997] and L<sup>2</sup>-penalty methods [Angot *et al., Num. Math.*, 1999]

The incompressible N-S equations with a penalized forcing term

$$\frac{\partial \mathbf{u}}{\partial t} + \nabla \cdot (\mathbf{u} \otimes \mathbf{u}) + \nabla P - \nu \nabla^2 \mathbf{u} = \mathbf{F} \text{ and } \nabla \cdot \mathbf{u} = \mathbf{0} \text{ in } \Omega$$
  
here  $\mathbf{F} = \frac{\alpha \chi_s}{\eta} (\mathbf{u}_i - \mathbf{u})$  with  $\alpha > 0, \ 0 < \eta \ll 1$  (here,  $\eta = 10^{-12}$ ) and  $\chi_s \to [0, 1]$ 

u<sub>i</sub> depending on u<sub>s</sub> and u following the requested accuracy

w

F is only applied on Cartesian nodes near/inside the immersed boundary (χ<sub>s</sub> > 0)

[Belliard, Fournier, *C.R. Acad. Sci. Paris*, 2010] & [Introini *et al.*, *Computers and Fluids*, 2014]

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wł

- **F** is only applied on Cartesian nodes near/inside the immersed boundary ( $\chi_s > 0$ )
- A consistent fractional step method [Ikeno & Kajishima, JCP, 2007] (prediction)  $\frac{\widetilde{\mathbf{u}} - \mathbf{u}^n}{\Delta t} + \nabla_h \cdot (\mathbf{u}^n \otimes \mathbf{u}^n) - \nu \nabla_h^2 \widetilde{\mathbf{u}} = \frac{\chi_s}{\eta \Delta t} (\mathbf{u}_l^{n+1} - \widetilde{\mathbf{u}})$  in  $\Omega$  with  $\alpha = 1/\Delta t$

 $\rightsquigarrow$  At this stage, IBCs are only satisfied by the provisional velocity  $\widetilde{\mathbf{u}}$ 

(correction) 
$$\frac{\mathbf{u}^{n+1} - \widetilde{\mathbf{u}}}{\Delta t} + \nabla_h P^{n+1} = \frac{\chi_s}{\eta \Delta t} (\widetilde{\mathbf{u}} - \mathbf{u}^{n+1}) \text{ in } \Omega$$

 $\rightarrow$  At the end of the projection stage, IBCs are also satisfied by the new velocity  $\mathbf{u}^{n+1}$ 

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#### The Penalized Direct Forcing method [Belliard, Fournier, C.R. Acad. Sci. Paris, 2010] & [Introini et al., Computers and Fluids, 2014]

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→ At the end of the projection stage, IBCs are also satisfied by the new velocity  $\mathbf{u}^{n+1}$ → Homogeneous Neumann BCs for *P* are naturally enforced on obstacles ( $\check{\rho}^{-1} \sim \mathcal{O}(\eta)$ )

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## Interpolation schemes $\underline{u}_i = \underline{u}_m(\underline{u}_S, \underline{u}^*)$

#### Base scheme

 $\blacksquare \underline{u_{\text{base}}}(\underline{x}) := \underline{u}_{\mathcal{S}}(\underline{x}) \quad \text{if } d(\underline{x}) \ge 0 \quad \diamondsuit$ 

d(<u>x</u>) signed distance to  $\Sigma_h$  (> 0 in  $\Omega_s$ )



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#### Linear interpolation scheme

$$\underline{u}_{\text{linear}}(\underline{x}) := \begin{cases} \underline{u}_{\text{lin_int}}(\underline{x}) & \text{if } -\Delta x \leq d(\underline{x}) < 0 \\ \underline{u}_{S}(\underline{x}) & \text{if } d(\underline{x}) \geq 0 \end{cases} \diamond$$

$$\Omega_I^{\text{linear}} = \{\diamondsuit, \blacklozenge\}$$



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But possible other choices (higher order schemes, extrapolations, ...)

$$\underline{u}_{\mathsf{m}}(\underline{x}) := \begin{cases} \underline{u}_{\mathsf{ho\_int}}(\underline{x}) & \text{if } -\Delta x \leq d(\underline{x}) < 0 \\ \underline{u}_{\mathsf{ho\_extra}}(\underline{x}) & \text{if } d(\underline{x}) \geq 0 \end{cases} \blacklozenge$$



#### In literature ...

- Volume-weighted model : Kajishima & al., IJ Heat and Fluid Flow 23 (2002)
- Differentially interpolated DF : Chiu & al., JCP 229 (2010)
- PDE/interpolation hybrid formulation : Luo & al., Comp. & Fluids 56 (2012)

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#### Regularization of $\underline{u}_i$ (Belliard & al. *in prep.*)

$$\begin{cases} \underline{\boldsymbol{u}}_{i} = \xi_{\mathsf{m}} \underline{\boldsymbol{u}}_{\mathsf{m}} + (1 - \xi_{\mathsf{m}}) \underline{\boldsymbol{u}}^{\star} \\ \underline{\boldsymbol{F}}_{RG} = \frac{\underline{\boldsymbol{u}}_{i} - \underline{\boldsymbol{u}}^{\star}}{\Delta t} \end{cases}$$

 $0 \leq \xi_m(\underline{x}) \leq 1$  : regular function of  $d(\underline{x})$ 

$$\underline{F}_{RG} = \xi_{\rm m} \frac{\underline{U}_{\rm m} - \underline{U}^{\star}}{\Delta t}$$

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📕 An example . . .

$$\xi_{\rm m}(\underline{x}) = \frac{\Delta x + 2[d(\underline{x}) - d_0^m]}{2\Delta x} \equiv \tau_{\rm m}(\underline{x})$$



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And also regul. hybrid PDF/DF method ...

1.2

1

0.8

0.4

χm

 $\Delta x$ 

τ<sub>m</sub> .....

## Penalized heat equation

The heat equation with a penalized forcing term

$$\partial_t T = \nabla \left( \frac{\lambda(T)}{\rho_0 C_{P0}} \overrightarrow{\nabla} T \right) \\ + \left( 1 - \chi_s \right) \frac{Q}{\rho_0 C_{P0}} \\ - \left( 1 - \chi_s \right) \overrightarrow{V} \cdot \overrightarrow{\nabla} T + \frac{\chi_s}{\eta} (T_i - T) \right)$$



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A fractional semi-implicit time scheme

$$\frac{\frac{T^* - T^n}{\delta t} - \nabla(\frac{\lambda(T^n)}{\rho_0 C_{P_0}} \overrightarrow{\nabla} T^*)}{\frac{T^{n+1} - T^*}{\delta t} - \frac{\chi_s}{\eta} (T_i - T^{n+1})} = (1 - \chi_s) \frac{Q}{\rho_0 C_{P_0}} - (1 - \chi_s) \overrightarrow{V^n} \cdot \overrightarrow{\nabla} T^n$$

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## Penalized heat equation

The heat equation with a penalized forcing term

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Models :

• Base : 
$$T_i = T_s$$

- Weighted :  $T_i = \tau_s T_s + (1 \tau_s) < T_{neig.} > \text{with } \tau_s|_K = vol_s|_K / vol_K$
- Treatment of fresh cells by an averaging process

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# Numerical applications



#### Poiseuille and Taylor-Couette flows







Rotating cylinder ( $\Re = 100, D/h \sim 50$ )

#### [Stojković *et al.*, Phys. Fluids, 02]<sup>1</sup> [Kang *et al.*, Phys. Fluids, 99]<sup>2</sup>

	Present	Ref.	
	results	[1]	[2]
$\overline{C_d}$	1.12	1.1080	1.0979
$C'_d$	$\pm 0.11$	$\pm 0.0986$	$\pm 0.0988$
$\overline{C_l}$	2.51	2.504	2.4833
$C'_{l}$	$\pm 0.37$	$\pm 0.3616$	$\pm 0.3603$
St	0.165	0.1658	0.1650



(solid/dotted lines : neg./pos. contours)



## 22 About the SFO damping

Analysis for the base model ...

$$\int_{\Omega_{s,h}^{n+1}} \underline{F}_{RDF}^{n+1} d\Omega - \int_{\Omega_{s,h}^{n}} \underline{F}_{RDF}^{n} d\Omega = \underbrace{\mathcal{O}(1) + \mathcal{O}(\Delta t) + \mathcal{O}(\Delta x)}_{K \in \Omega_{s,h}^{n+1} \cap \Omega_{s,h}^{n}} + \underbrace{\mathcal{O}(\Delta t) + \mathcal{O}(\Delta x)}_{K \in DC} + \underbrace{\mathcal{O}(\Delta t)}_{K \in FC} + \underbrace{\mathcal{O}(\Delta t)}_$$

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## About the SFO damping

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An oscillating cylinder in cross-flow ( $R_e = 185$ , Guilmineau & Queutey, JFS 16, 2002)



An oscillating cylinder in cross-flow ( $R_e = 185$ , Guilmineau & Queutey, *JFS 16, 2002*)



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## Cea 2D molten glass flow over a cold cylinder

- $\mu(T) \in [1; 10^5]$ ,  $\Lambda(T)$  and Boussinesq approx.
- Fluent benchmark simulation (boundary layer : AMR ; 14,000 faces)
- Mesh convergence study (2,400 → 10 M cells)



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D0: post\_calnete\_setoid.ioto

Mesh convergence study (2,400  $\rightarrow$  10 M cells)







## Cold crucible simulations

#### Hydraulic validations of flows induced by a stirrer

- Experimental device filled with viscous oil ( $\Re \approx 100$ ) PIV -
- IBC Lagrangian meshes from CAD
- Regular Eulerian meshes



## Cold crucible simulations

#### Hydraulic validations of flows induced by a stirrer

- Experimental device filled with viscous oil ( $\Re \approx 100$ ) PIV -
- IBC Lagrangian meshes from CAD
- Regular Eulerian meshes
- Velocity-profile comparisons on a cut-plane and lines
- Fluent body-fitted sliding-mesh simulations (1 M cells with AMR)







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#### **Base model** : mesh convergence study ( $0.05 \rightarrow 20$ M cells ; $2 \rightarrow 400$ CPUs)





Experimental rate of convergence about 0.6

**Base model** : mesh convergence study ( $0.05 \rightarrow 20$  M cells ;  $2 \rightarrow 400$  CPUs)





Experimental rate of convergence about 0.6

Linear vs base model : cost  $\searrow$  for a given accuracy (1 M cells is quasi-optimal)



Experimental rate of convergence about 1.0 for coarse meshes (but, exp. errors...)
Trio\_U results compare well with experiment and Fluent ones

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## Cea Cold crucible simulations

#### **Thermal-hydraulic** simulations ; $\mu(T) \in [1; 10^5]$ , $\Lambda(T)$ and Boussinesq approx., $\Re \approx 100$



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## Cea Cold crucible simulations

#### Thermal-hydraulic simulations ; $\mu(T) \in [1; 10^5]$ , $\Lambda(T)$ and Boussinesq approx., $\Re \approx 100$



Hydraulic bubbling simulations



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Conclusions -Perspectives

## Conclusions and perspectives

#### Development of a Penalized Direct Forcing method

- Finite Volume approximation with a staggered arrangement of variables
- To solve the Navier-Stokes equations on Cartesian grid with penalized forcing term taking into account immersed boundary conditions
- $\rightsquigarrow$  To account for the IBCs in the prediction and correction equations
- Development of an original robust interpolation scheme
  - → To reconstruct accuratly the velocity field near the immersed boundary
- SFO analysis & development of regularized DF and PDF methods
- **CFD assessment** on Academical test cases ( $\mathcal{O}(h^2)$  in  $L^2$  and  $L^{\infty}$ -norms)

Cutting off the SFO without altering physical phenomena nor the space convergence orders

- Numerical simulations of the cold crucible process
  - → Hydraulic assessment vs experiment and code results
  - → Thermal-hydraulic assessment vs code benchmark
  - → Hydraulic bubbling simulations with IBCs give satisfactory results



#### $\implies$ Increase of the validation basis

#### → Extension of our Penalized Direct Forcing method

- Energy equation : linear interpolation for the temperature
- Including AMR ...
- Increasing the Reynolds number ...

- $\implies$  FSI applications for single-phase and two-phase fluids
- $\implies$  Cold crucible process simulations
  - Bubbling : convergence study and assessment vs experiment

## Thank you for your attention ...

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